

## FORMULAS

### FAN, PUMPS AND DRIVES

$$\frac{BHP1}{BHP2} = \left(\frac{CFM1}{CFM2}\right)^2 = \left(\frac{RPM1}{RPM2}\right)^2 = \left(\frac{SP1}{SP2}\right)^3 \quad \text{Affinity Laws, Note: (CFM = GPM)}$$

$$\text{Synchronous Motor RPM} = \frac{\text{Hertz} \times 120}{\text{Poles}}$$

$$\text{Belt\_Length} = 2c + [1.57 \times (D + d)] + \frac{(D-d)^2}{4c}$$

$$\frac{\text{RPM Motor}}{\text{RPM Fan}} = \frac{\text{DIA Fan Sheave}}{\text{DAI Motor Sheaves}}$$

$$BHP_{\text{Actual}} = 1.73 \times \text{Amps} \times \text{Volts} \times \eta_{\text{motor}} \times \text{P.F.}$$

$$\text{Rule of Thumb: } BHP_{\text{Actual}} = \text{HP}_{(\text{name\_plate})} \times \frac{\text{Amps Actual}}{\text{Amps Rated}} \times \frac{\text{Volts Actual}}{\text{Volts Rated}}$$

$$BHP = \frac{\text{GPM} \times \text{FTHD} \times \text{S.G.}}{6,350 \times \eta_{\text{pump}}} \quad (\text{For Pumps})$$

$$BHP = \frac{\text{CFM} \times \text{in. W.G.}}{6,350 \times \eta_{\text{fan}}} \quad (\text{For Fans})$$

### ELECTRICAL EQUATIONS

$$\text{KVA} = \frac{\text{Amps} \times \text{Volts} \times \sqrt{Ph}}{1,000}$$

$$\text{KW} = \text{KVA} \times \text{P.F.} = \frac{\text{P.F.} \times \text{Amps} \times \text{Volts} \times \sqrt{Ph}}{1,000}$$

$$\text{KW motor input} = \frac{\text{BHP} \times 0.746}{\eta_{\text{motor}}}$$

$$V = IR$$

$$W = V \times I = I^2 \times R$$

$$\text{KW}_{\text{dc}} = \frac{\text{Amps} \times \text{Volts}}{1,000}$$

### REFRIGERATION

$$\text{KW / ton} = 12 / \text{EER}$$

$$\text{COP} = \text{EER} / 3.412 = 12 / (\text{KW / ton}) / 3.412$$

$$\text{Cond. Tons} = \frac{\text{GPM} \times \Delta T \times (1 + \frac{1}{\text{COP}})}{24}$$

### WATER SIDE HVAC FORMULAS

$$\text{BTUH} = \text{GPM} \times 500 \times T \text{ (water)}$$

$$\text{TONS} = \frac{\text{GPM} \times \Delta T}{24} \text{ (CH water) (CT Ton} = 15,000 \text{ BTUH)}$$

$$\text{FTHD} = \frac{\text{psi} \times 2.31}{\text{S.G.}}$$

$$\text{NPSH}_A = h_a - h_{\text{vpa}} + h_{\text{st}} - h_{\text{fs}}$$

$$h_a = \text{Absolute Pressure in feet of liquid on surface supply level.}$$

$$h_{\text{vpa}} = \text{Head in feet corresponding to vapor pressure of liquid at the temperature being pumped.}$$

$$h_{\text{st}} = \text{Static height that the liquid level is above (+) or below (-) the pump centerline}$$

$$h_{\text{fs}} = \text{All suction line losses including the entrance loss and friction losses through pipe, valves and fittings.}$$

### AIR SIDE HVAC FORMULAS

$$\text{BTUH}_{\text{Total}} = \text{BTUH}_{\text{Sensible}} + \text{BTUH}_{\text{Latent}}$$

$$\text{BTUH}_{\text{Sensible}} = (1.08) \times (\text{CFM}) \times \Delta T$$

$$\text{BTUH}_{\text{Total}} = (4.5) \times (\text{CFM}) \times \Delta H$$

$$\text{ACH} = \frac{60 \times \text{CFM}}{\text{Floor Area} \times \text{Ceiling Height}}$$

$$P_{\text{velocity}} = \left(\frac{V}{4,005}\right)^2 = \left(\frac{V}{C}\right)^2 \left(\frac{P}{2g}\right) \quad C = 136.8, g = 32.2$$

$$P_{\text{Total}} = P_{\text{Velocity}} + P_{\text{Static}}$$